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PATENT

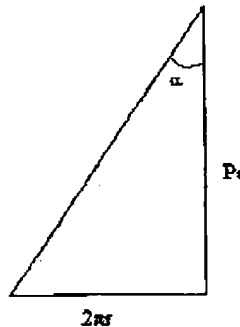
KOD177B.001APC

Applicant	: Matsuda, et al.
Appl. No.	: 10/595,804
Filed	: June 12, 2006

I. Clarification is required on the interpretation of Osako (6220983) with respect to the angle of twist.

- The angle of twist α can be calculated as follows:

$\alpha = \tan^{-1}(2\pi/P_s)$, wherein r is a spiral radius, P_s is a twisting pitch.



In Osako, the diameter ($2r$) is 1.1 to 1.5 mm, resulting in $2\pi r = 3.46$ to 4.71 mm. The final twist number is 5 to 10 twists/10 cm, resulting in $P_s = 10$ to 20 mm. Thus, $\alpha = \tan^{-1}((3.46 \text{ to } 4.71)/(10 \text{ to } 20)) = \tan^{-1}(0.173 \text{ to } 0.471) = 9.8$ to 25.2° .

II. The differences between Umezawa (5520233) and the claimed invention need to be discussed.

- The structure (a heavy duty pneumatic tire) requires:
 - at least one carcass ply containing cords;
 - a first belt layer (containing cords inclined at a large cord angle, e.g., 50° , with respect to the equatorial plane of the tire to develop a "propping effect", enhancing the tensile rigidity of the belt); and
 - at least two belt layers (each containing cords inclined at a small cord angle, e.g., 20° , with respect to the equatorial plane), in which cords of the second and third belt layers are crossed with each other.

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- The tensile rigidity of the belt is evaluated by the resistance to cord breakage when the internal pressure applied to the tire is reciprocally changed from 1.0 kgf/cm² to 7.0 kgf/cm² (the shoulder portion of the tire is continuously subjected to input from projections existing on road surface) (col. 6).
- There is no motivation to single out the cords from the above complex structure and place them in a helical synchronous belt (the core cord twist angle is an angle with respect to the longitudinal direction of the belt).